

# PROTOCOL FOR HEAVY DUTY HYDROGEN REFUELING: A MODELING BENCHMARK

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Horizon 2020  
European Union Funding  
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# 1

## Introduction

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# 1) Introduction

## Modeling software for hydrogen fillings

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- Use of hydrogen in mobility uses (road transport, ...) to increase
- Hydrogen Refueling Stations (HRS) needed to be safe, fast and easy-to-use
- Gaseous filling → heat generation in the vehicle tanks
- Filling protocol describes how the HRS should behave
- Development of protocol dependent on modeling

Accuracy of simplified model is very important

PRHYDE project: focus on Heavy Duty (trucks)

# 2

## Modelled cases

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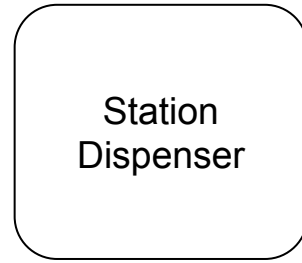
## 2) Modelled cases

10 minutes filling

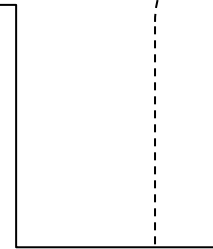
$T_{amb} = 15\text{ }^{\circ}\text{C}$

**Input** Dispenser conditions (Gas T / P)

	H35	H50	H70
Initial pressure (MPa)	6	8	10
End dispenser pressure (MPa)	43.75	62.5	87.5
Dispenser temperature ( $^{\circ}\text{C}$ )	15	5	-10
Reference mass flow (for pressure drop, g/s)	84	111	141

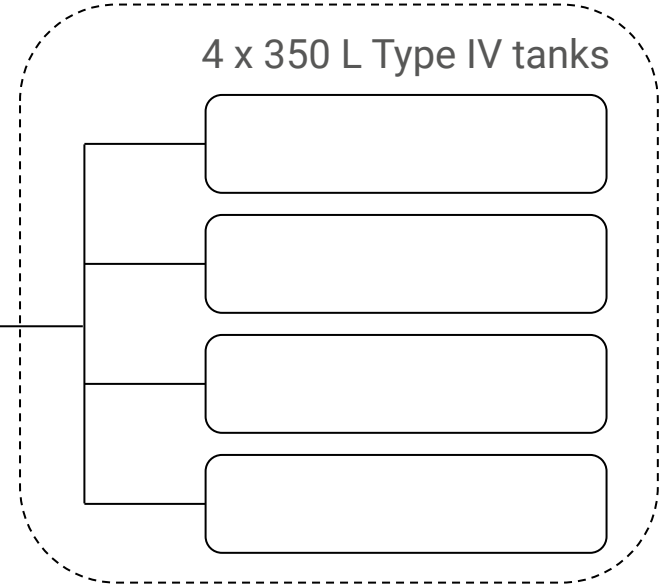


Piping



Vehicle

4 x 350 L Type IV tanks



**Output** Tank Gas T / P

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# 3

## Models descriptions

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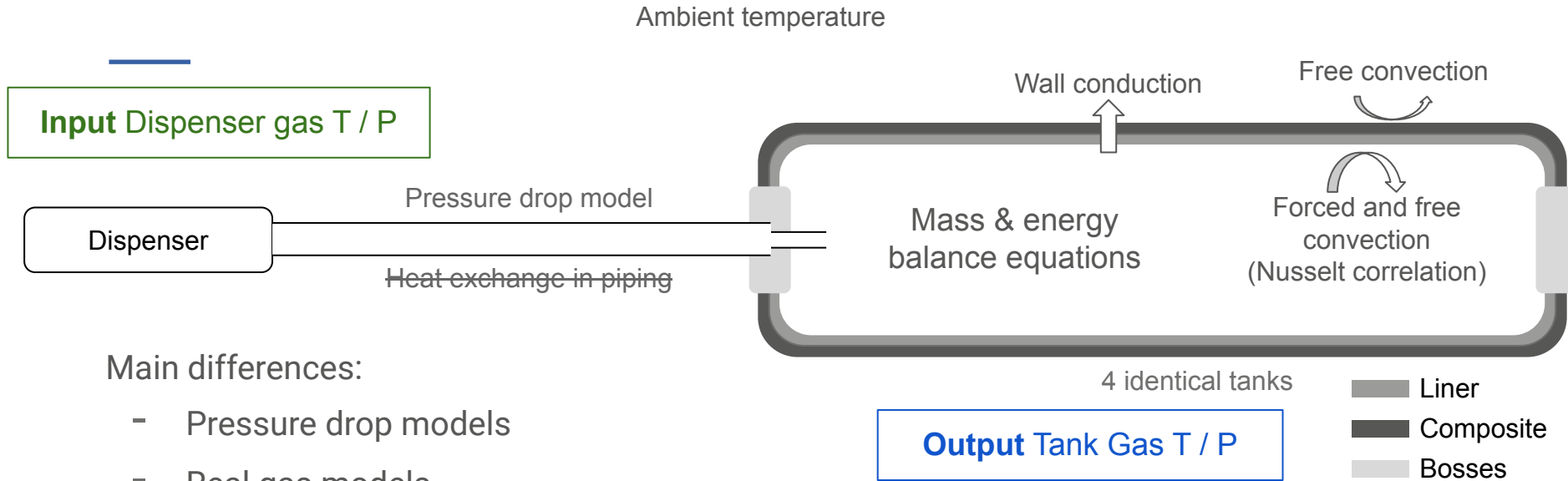


### 3) Models descriptions

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- **Air Liquide** SOFIL [1, 2]
- **Engie** Hyfill [3-6]
- **NREL** H2FillS [7, 8]
- **Wenger Engineering** H2-Fill [9, 10]

### 3) Models descriptions



Main differences:

- Pressure drop models
- Real gas models
- 1D-Wall conduction (radial: Air Liquide, Engie, Wenger / cartesian: NREL)
- External heat transfer coefficient  
(fixed: Engie, NREL / Nusselt correlation: Air Liquide, Wenger)



# 4

## Benchmark

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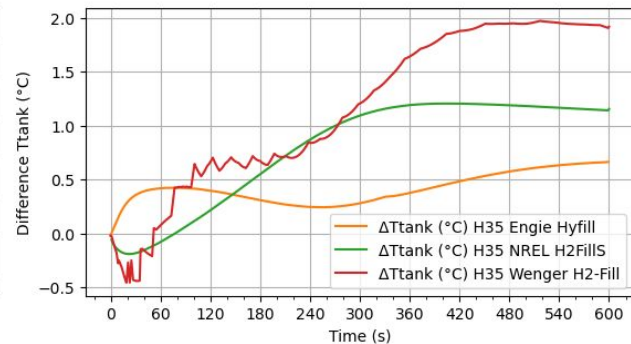
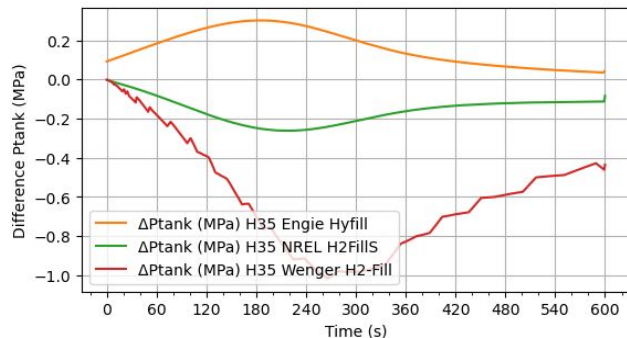
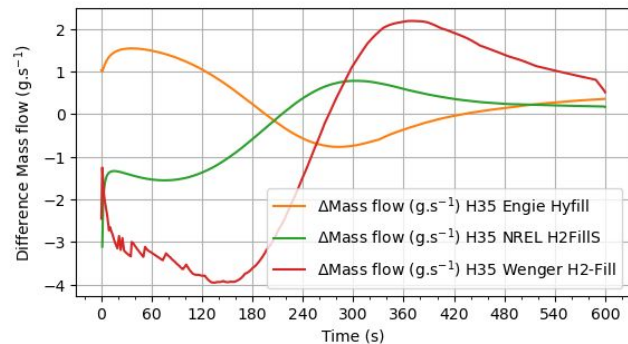
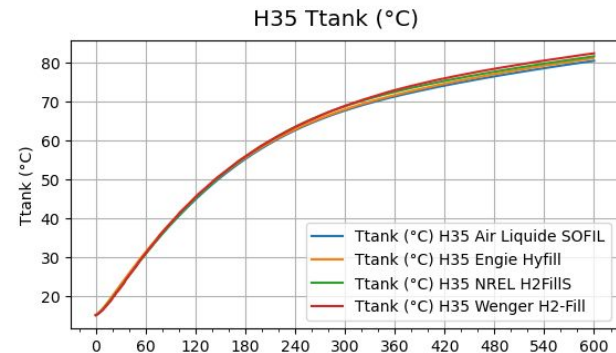
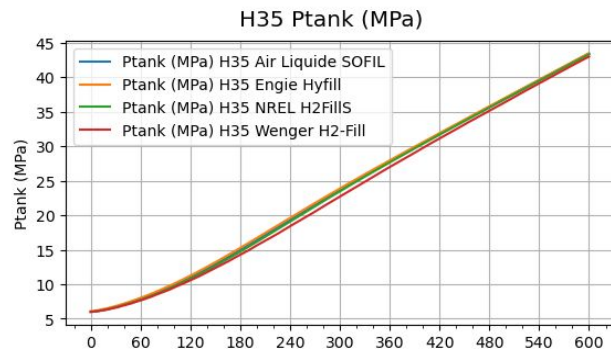
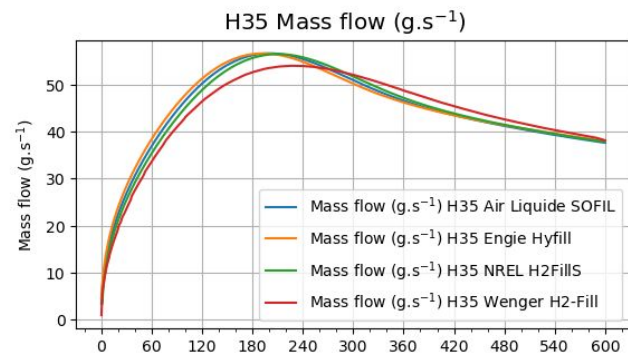
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# 4) Benchmark - H35



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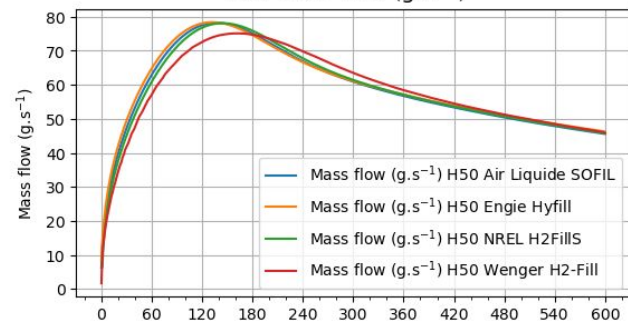
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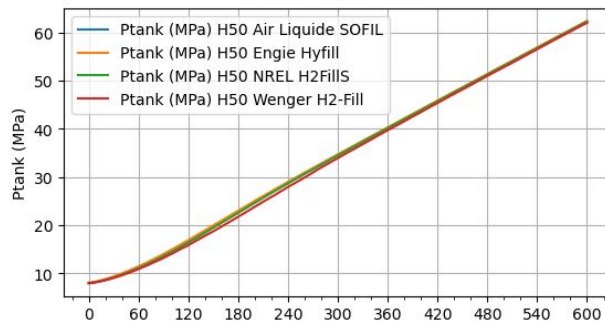


# 4) Benchmark - H50

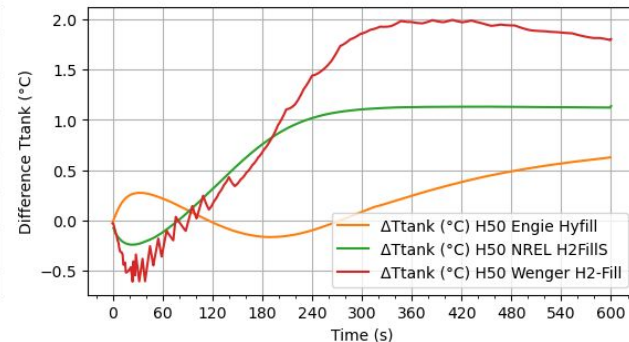
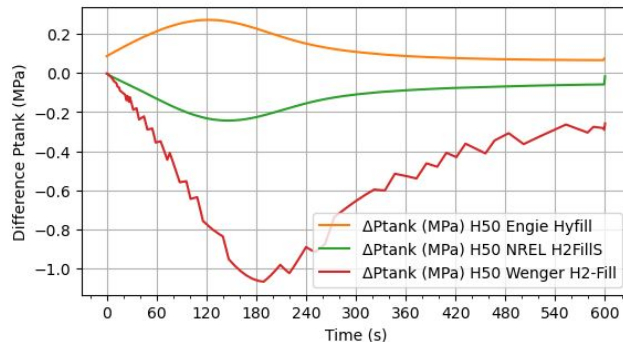
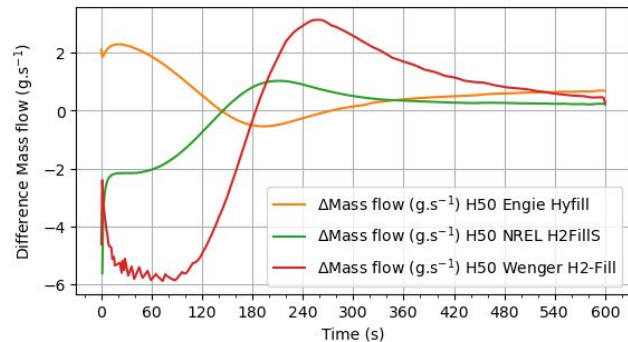
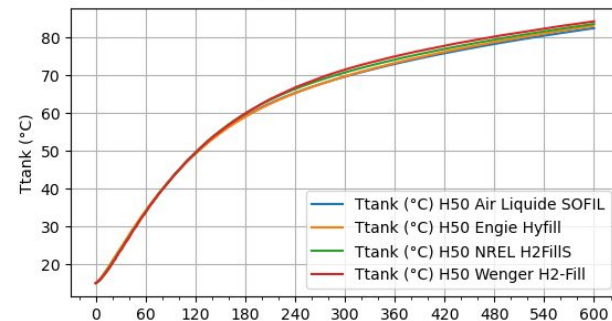
H50 Mass flow ( $\text{g.s}^{-1}$ )



H50 P<sub>tank</sub> (MPa)

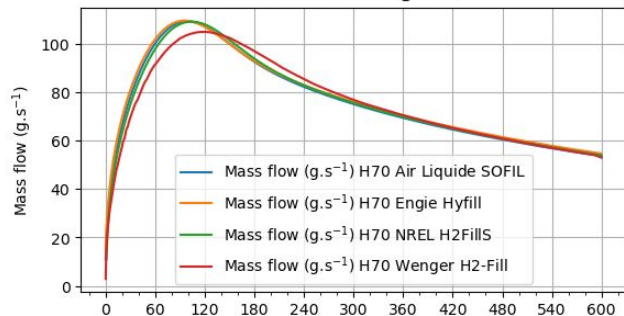


H50 T<sub>tank</sub> (°C)

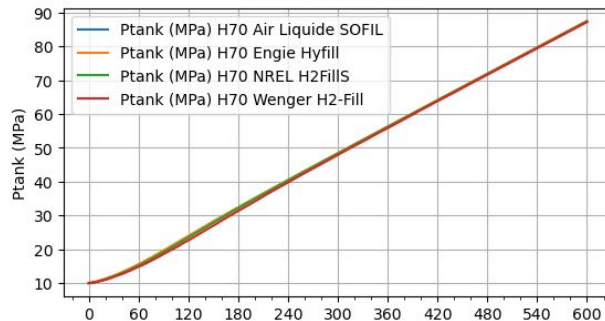


# 4) Benchmark - H70

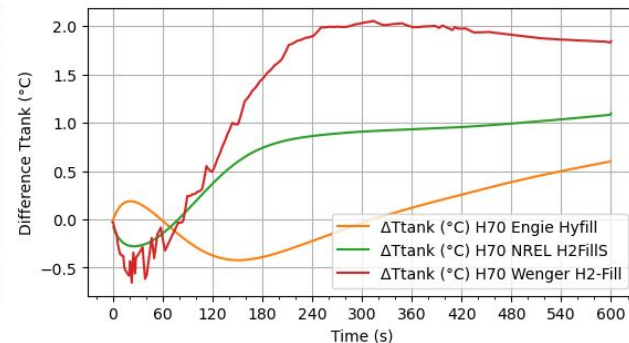
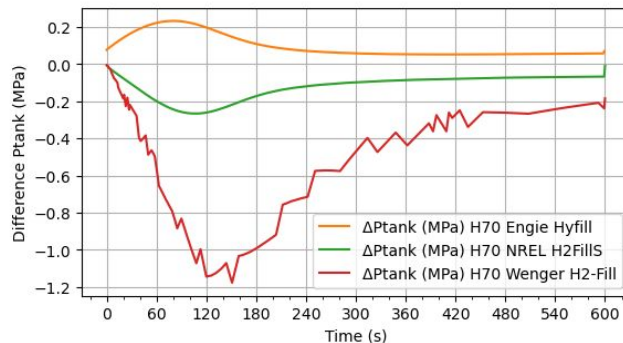
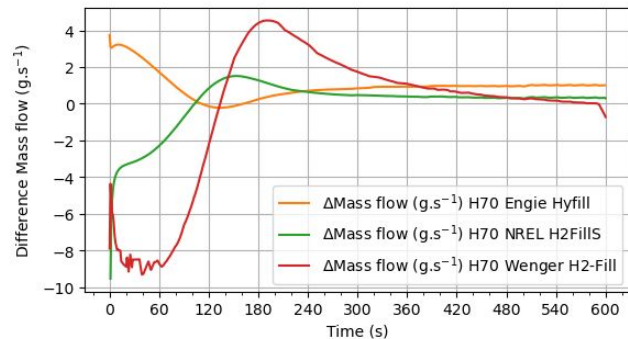
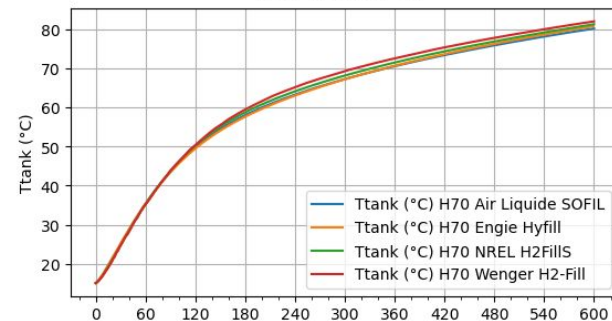
H70 Mass flow ( $\text{g.s}^{-1}$ )



H70 Ptank (MPa)



H70 Ttank ( $^{\circ}\text{C}$ )



# 5

## Conclusion

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## 5) Conclusion

- Very similar predictions: less than 2 °C range
- Differences:
  - Real gas equations
  - Bosses modeling
  - Simplified tank geometry implementations
  - Pressure drop formula → mass flow

**Models used for protocol development in PRHYDE.**

## Acknowledgment

Fuel Cells and Hydrogen 2 Joint Undertaking: Grant Agreement  
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# Annex

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# A) Pressure drop models\*

Air Liquide's SOFIL

Sonic flow ( $P_1 > 2 P_2$ )

$$\frac{dm_g}{dt} = C k_v P_1 \sqrt{\frac{\rho_N}{T_1}}$$

Subsonic flow ( $P_1 \leq 2 P_2$ )

$$\frac{dm_g}{dt} = 2 C k_v \sqrt{\frac{\rho_N (P_1 - P_2) P_2}{T_1}}$$

Engie's Hyfill

Sonic flow ( $P_1 > 2 P_2$ )

$$\frac{dm_g}{dt} = \rho_1 N k_v Y \sqrt{\frac{P_1}{2\rho_1}} \text{ with } Y = \frac{2}{3}$$

Subsonic flow ( $P_1 \leq 2 P_2$ )

$$\frac{dm_g}{dt} = \rho_1 N k_v Y \sqrt{\frac{(P_1 - P_2)}{\rho_1}}$$

with  $Y = 1 - \frac{2}{3} \frac{P_1 - P_2}{P_1}$

NREL's H2FillS

Sonic flow ( $P_1 > 2 P_2$ )

$$\dot{V} = 2930 C_v \sqrt{\frac{(P_1 - P_2)(P_1 + P_2)}{P_1 G T_1}}$$

Subsonic flow ( $P_1 \leq 2 P_2$ )

$$\dot{V} = 2538 C_v \frac{P_1}{G T_2}$$

Conversion to mass flow

$$\dot{m} = \frac{\beta \rho \dot{V}}{3600}$$

$\beta$  coefficient  $\rightarrow$  handle unsteady flow

\* Wenger model not described



# B) References

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